

Experiments and computations to address the safety case of heat pipe failures in Special Purpose Reactors

(NEUP Project 19-17416)

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Huang, Taehwan Ahn

Introduction

- Special Purpose Reactor
- Operation of heat pipe

- Viscous limit

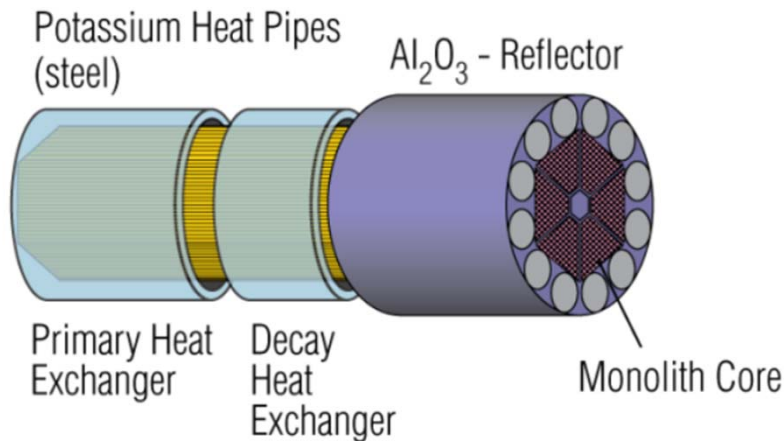
$$\bar{q}_v = \frac{d^2 h_{fg}}{64 \mu_v l_{eff}} \left(1 - \frac{P_1^2}{P_0^2} \right) \rho_0 P_0$$

- Sonic limit

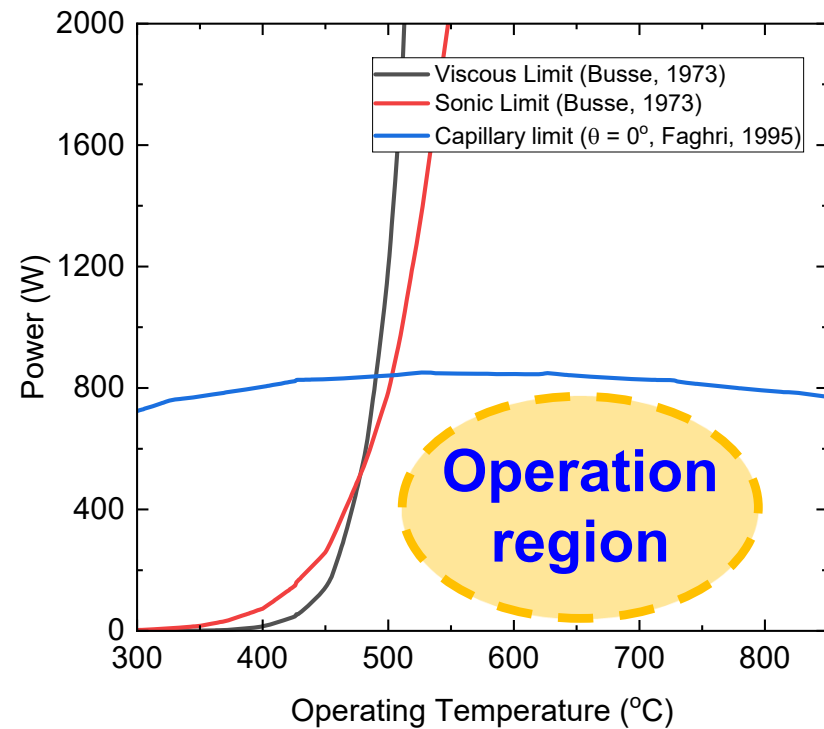
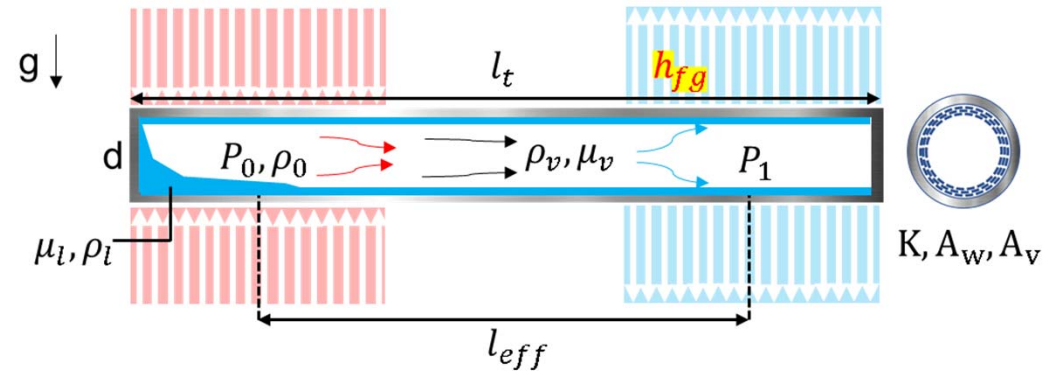
$$\bar{q}_s = 0.474 h_{fg} \sqrt{\rho_0 P_0}$$

- Capillary limit

$$\bar{q}_c = \frac{\sigma_l \rho_l h_{fg} K A_w}{\mu_l l_{eff}} \left(\frac{2}{r_{eff}} - \frac{\rho_l g l_t \cos \theta}{\sigma_l} \right)$$

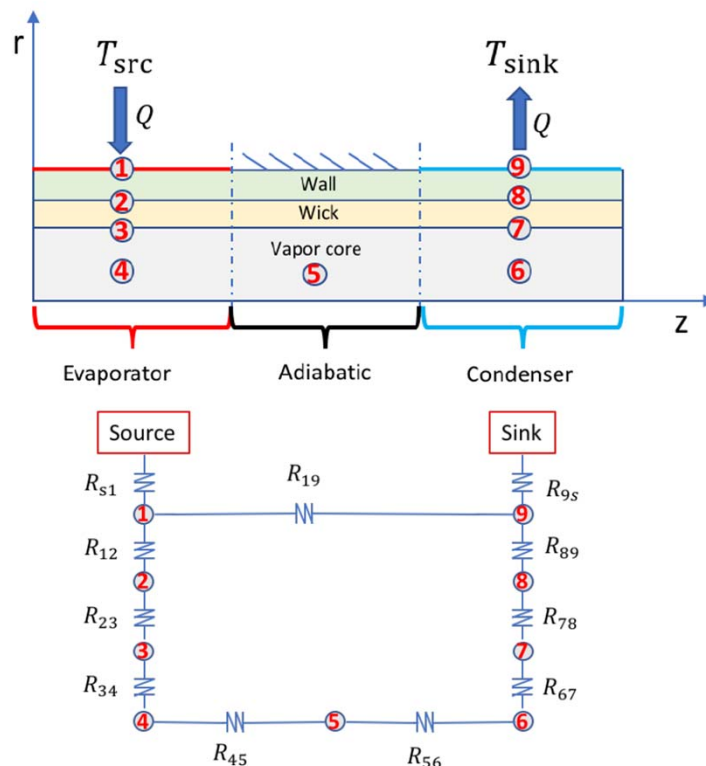


Special Purpose Reactor

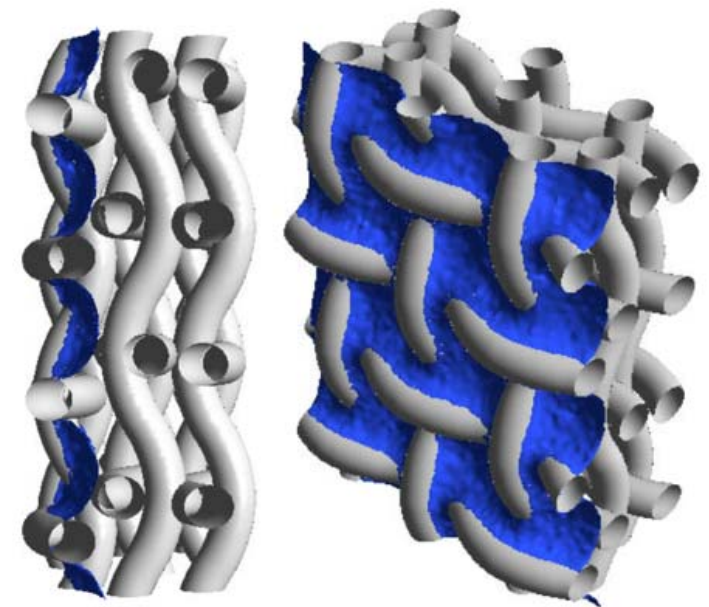


Introduction

- Issues remaining for the licensing of microreactors
 - The heat pipe modeling is either over-simplified or too expensive
- Knowledge gaps
 - The two-phase flow phenomena in a heat pipe
 - The separate effects of a heat pipe
 - The integral study of heat pipes bundle



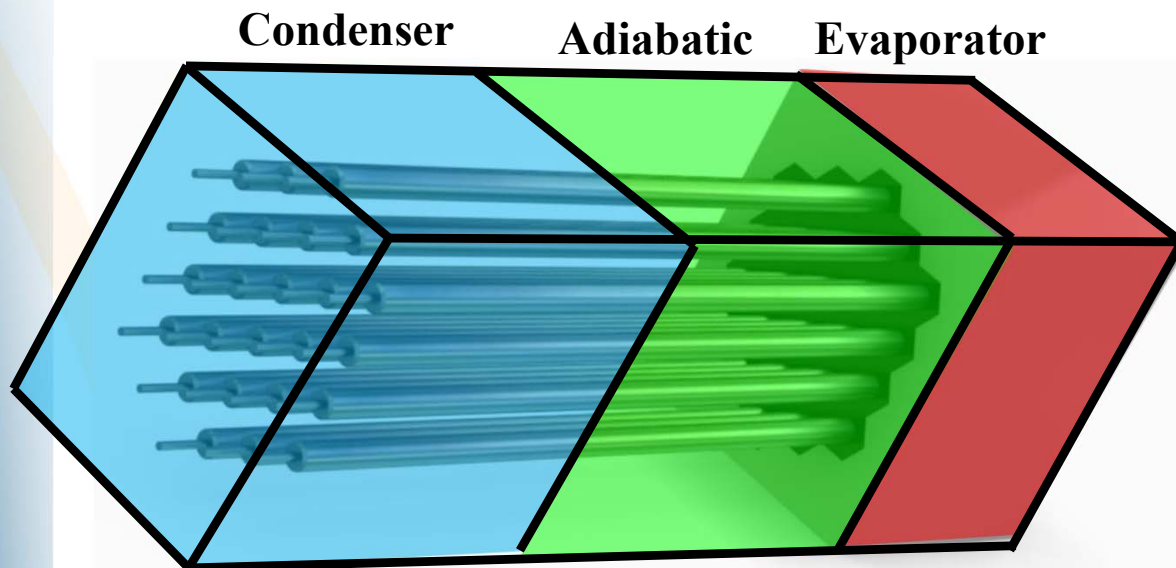
(Hu et al., 2019)



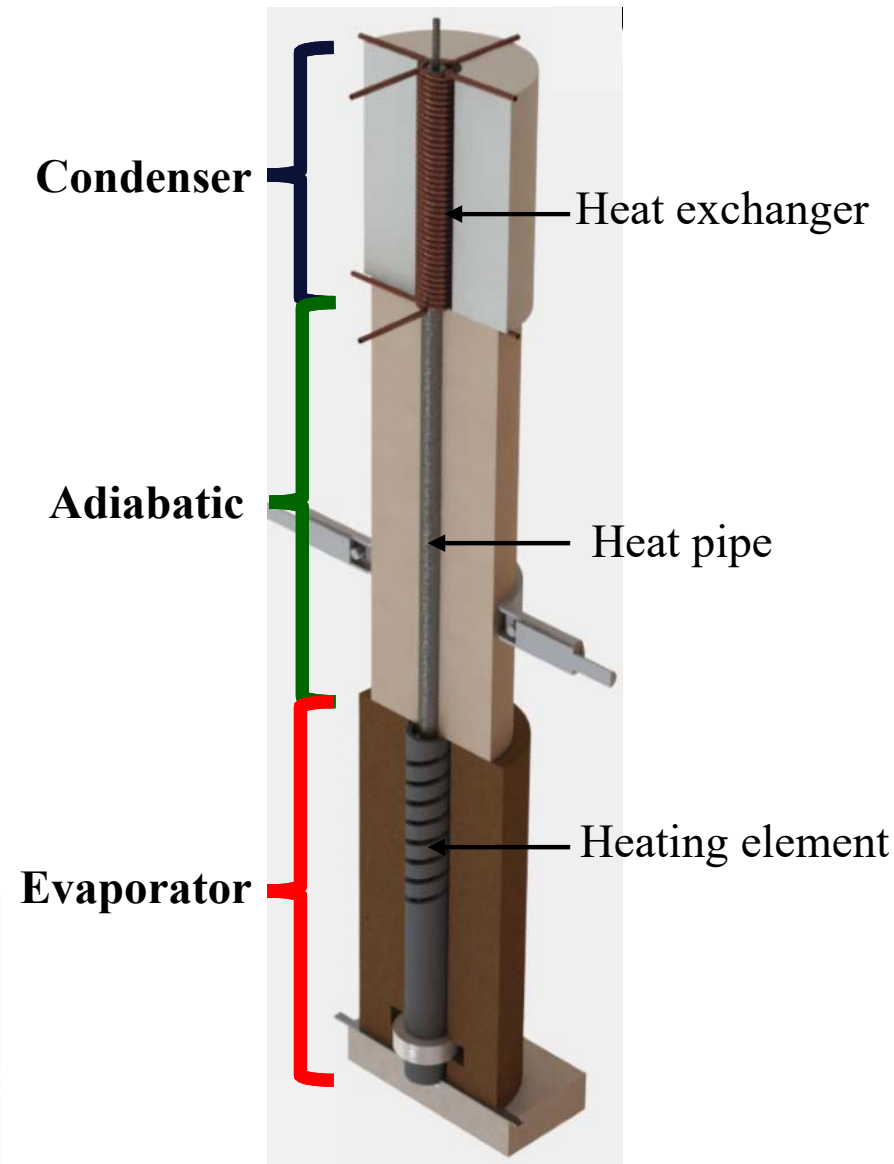
<Volume of Fluid (VOF) model>
(Dutra et al., 2022)

Project goal

- Separate effect of sodium heat pipe
 - Visualization using X-ray radiography
 - Key parameter investigation
- Integral effect of heat pipes bundle
 - Operation under various conditions



<Test facility for integral effect of sodium heat pipe bundle>



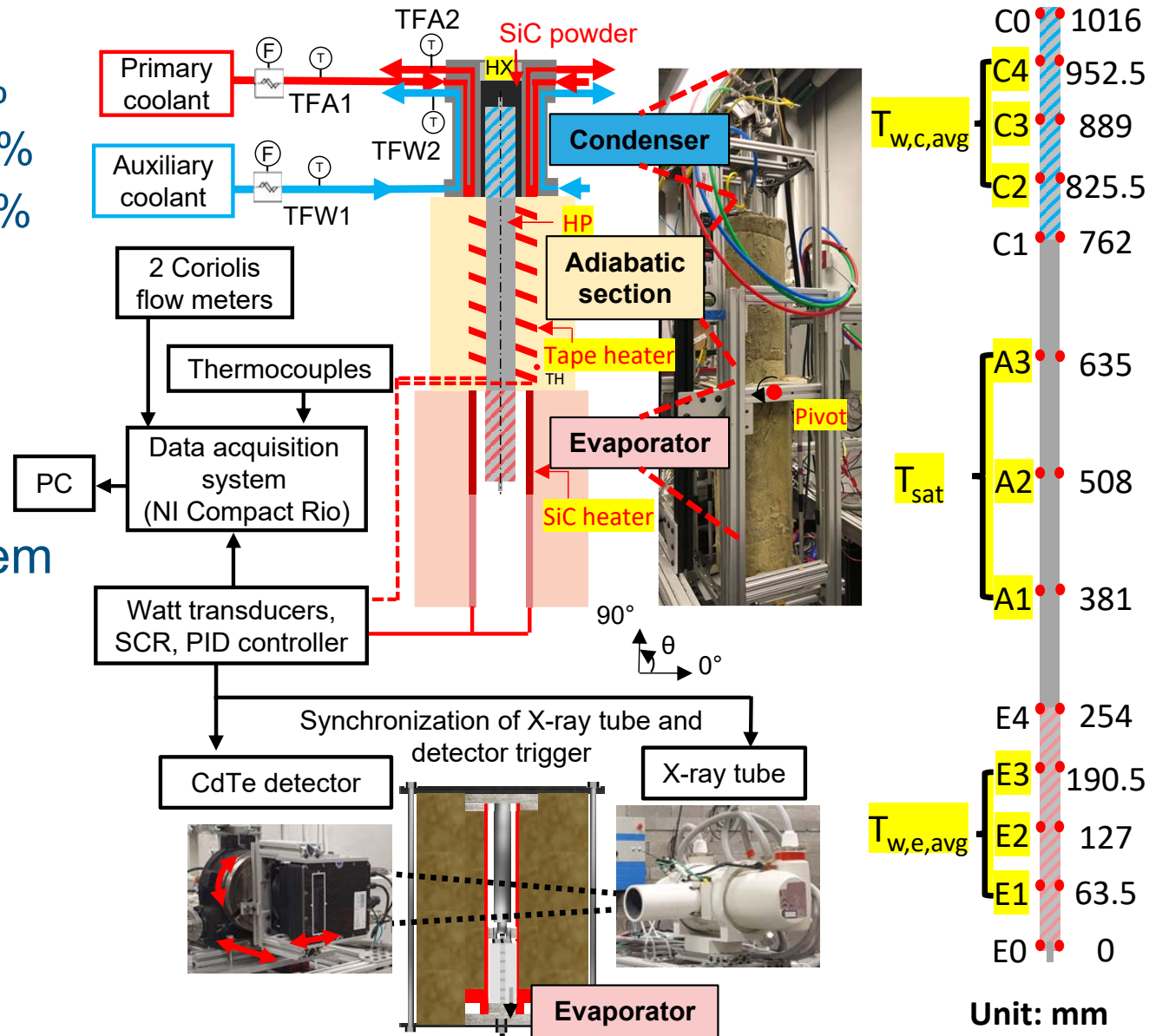
<Test facility for separate effect of single sodium heat pipe>

1. Single heat pipe tests

Flow phenomenon and separate effect investigation with Michigan Sodium Heat pipe test facility (MISOH1)

Experimental apparatus

- Sodium heat pipe
 - HP-a filling ratio = 67%
 - HP-b filling ratio = 102%
 - HP-c filling ratio = 172%
- MISOH1 test facility
 - Evaporator
 - Adiabatic
 - Condenser
- X-ray radiography system
 - X-ray generator
 - Thor detector
 - Triggering module



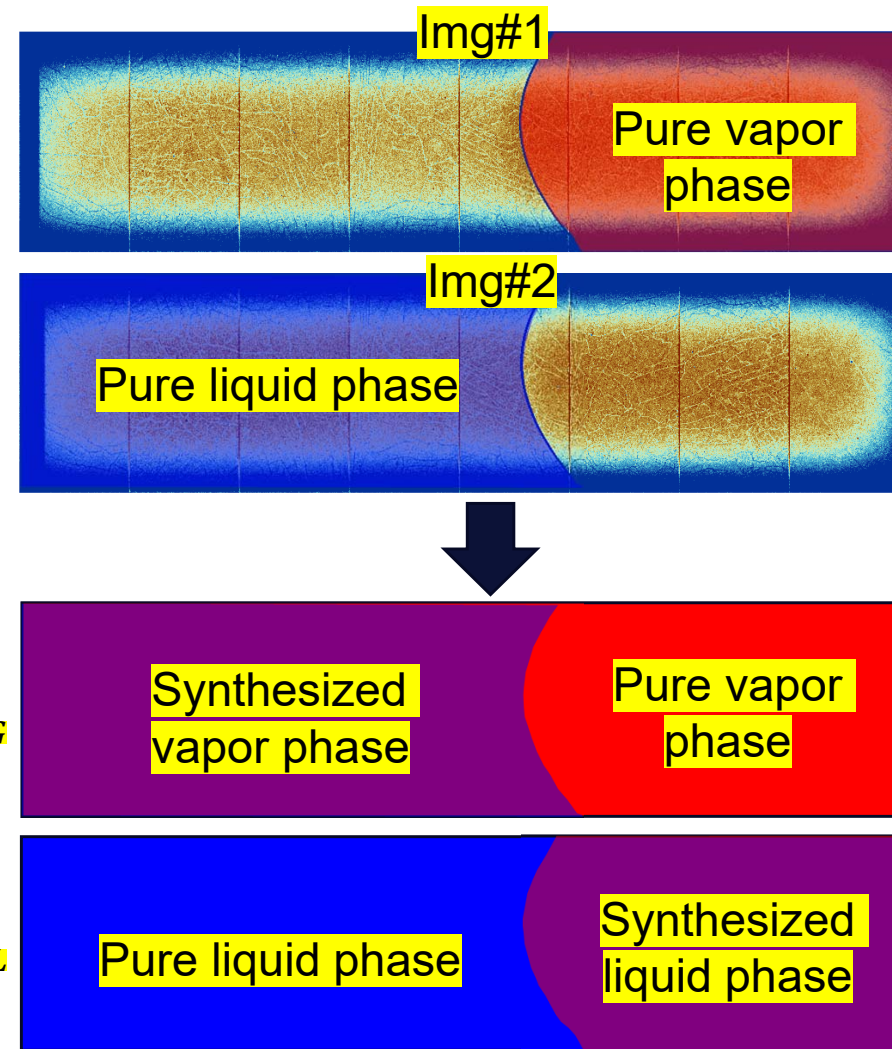
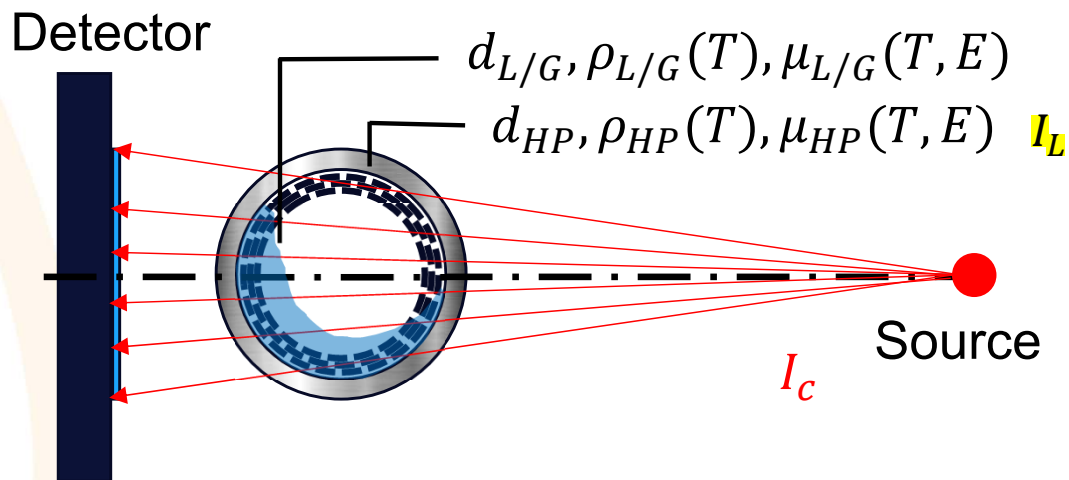
Experimental apparatus

- Data processing (X-ray)
 - Pure liquid/vapor intensity ($I_{L/G}$):

$$I_{L/G}(T) = I_c e^{-\rho_{HP}(T)\mu_{HP}(E)d - \rho_{L/G}(T)\mu(E)d}$$

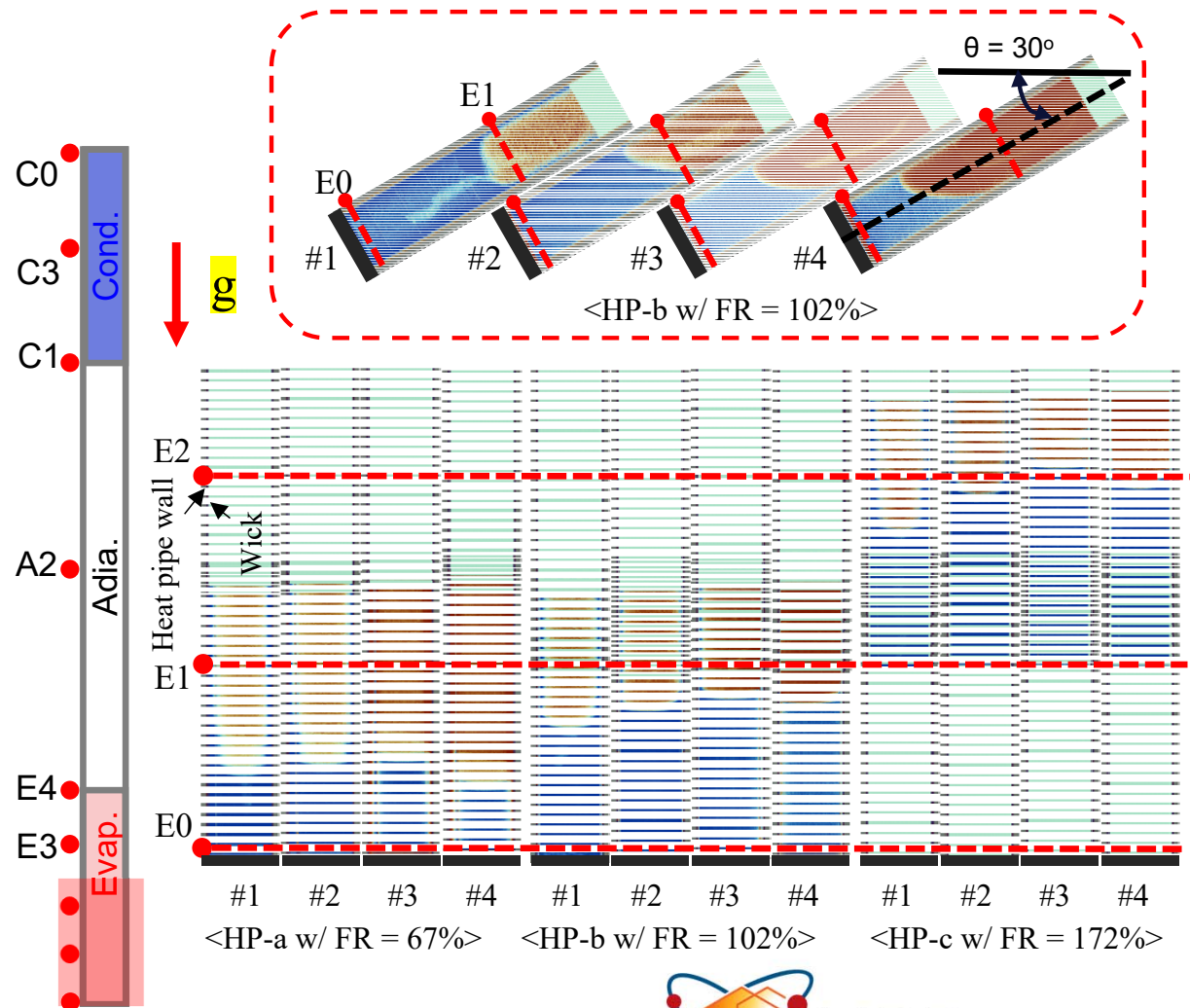
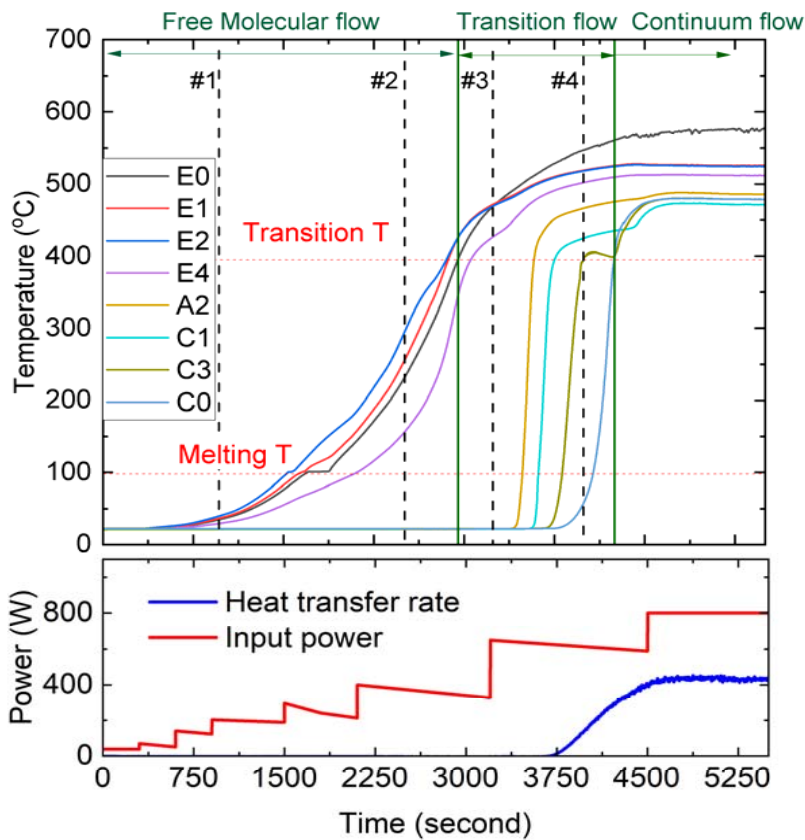
- Normalized attenuation value (I_M):

$$I_M = \frac{\ln(I) - \ln(I_L)}{\ln(I_G) - \ln(I_L)}$$



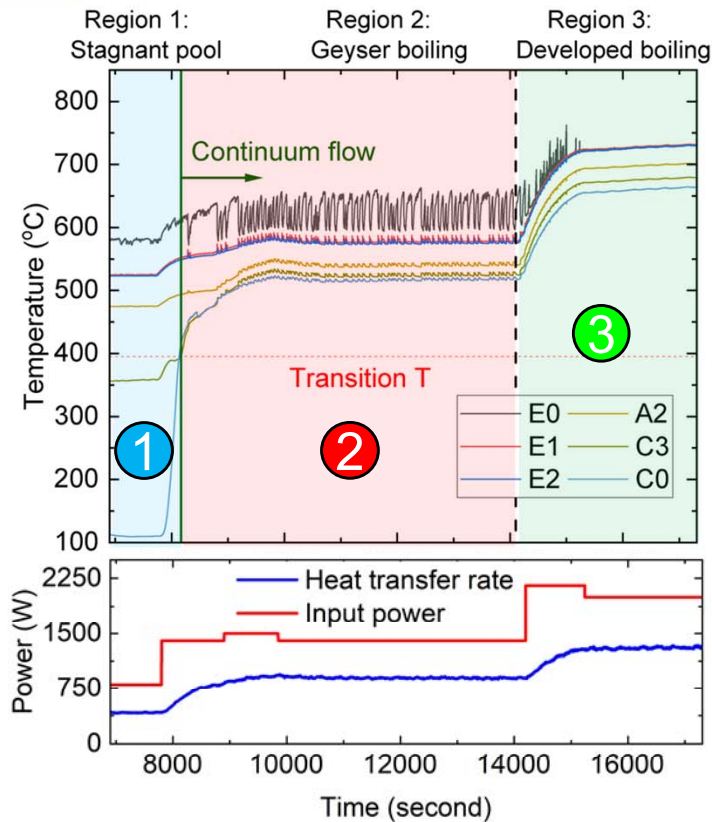
Result 1: Heat pipe transient characteristics

- Heat pipe experiences various flow regimes during startup process
- The visualization of heat pipes with different filling ratios and inclination angles



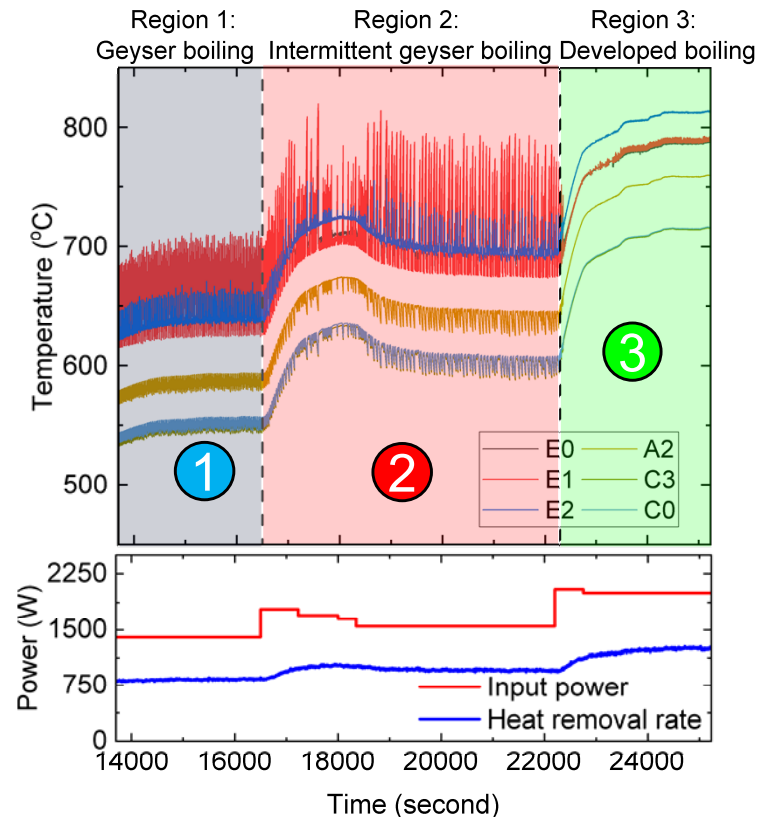
Result 1: Heat pipe transient characteristics

- Heat pipes experience two-phase flow phenomenon varies with the operation conditions



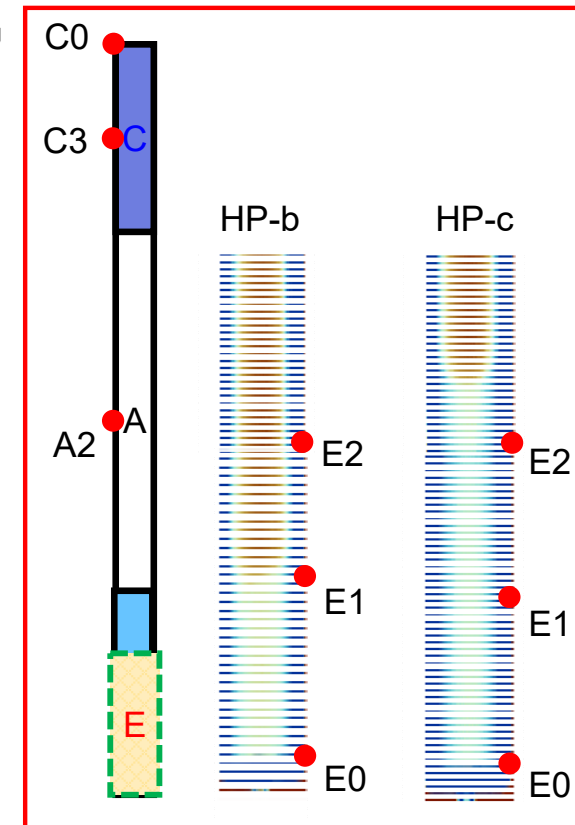
HP-b (102% FR):

- Vertical orientation (90°)
- Cooling air mass flow rate: 7 g/s



HP-c (172% FR):

- Vertical orientation (90°)
- Cooling air mass flow rate: 4 g/s

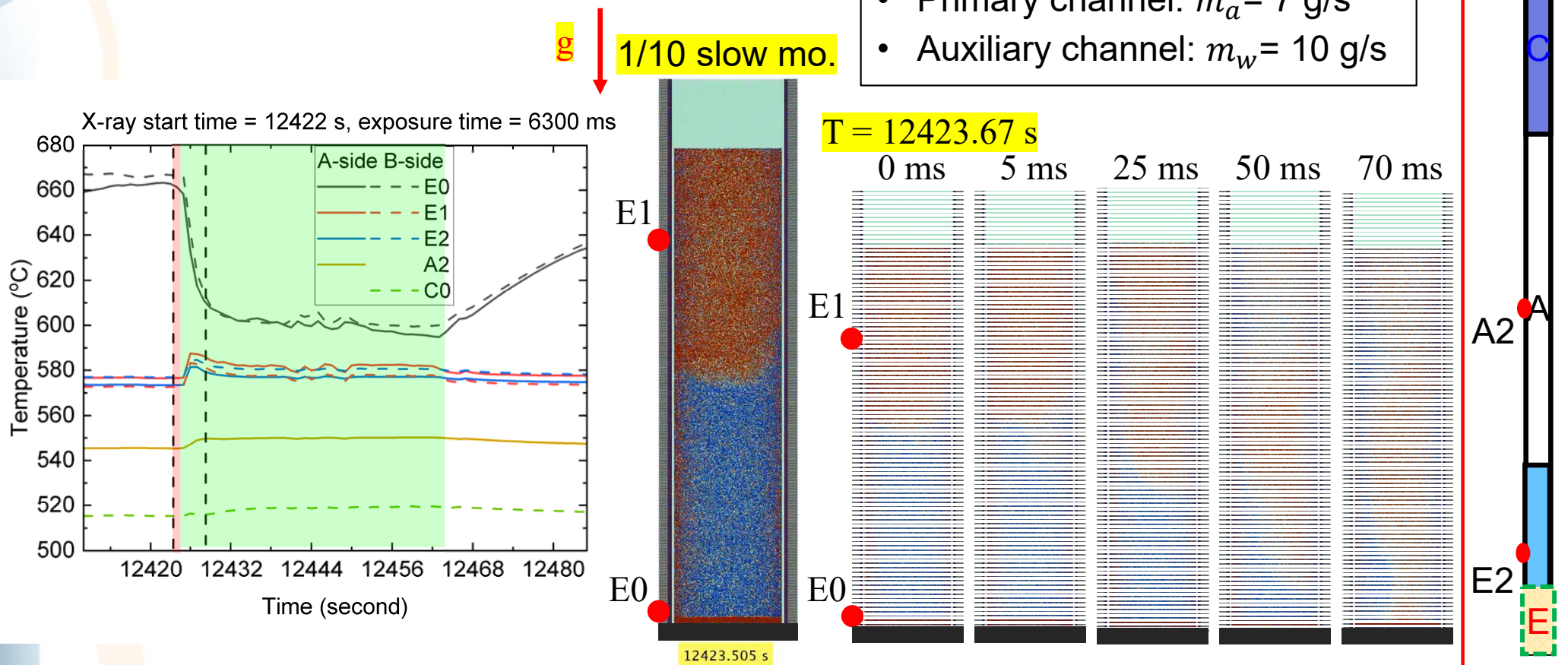


Result 2: Flow phenomenon

- Sequence of geyser boiling phenomenon video synchronized with temperature data

HP-b (102% FR):

- Vertical orientation (90°)
- Primary channel: $m_a = 7$ g/s
- Auxiliary channel: $m_w = 10$ g/s



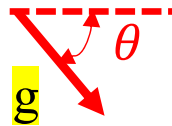
Result 2: Flow phenomenon

- Sequence of geyser boiling phenomenon video synchronized with temperature data

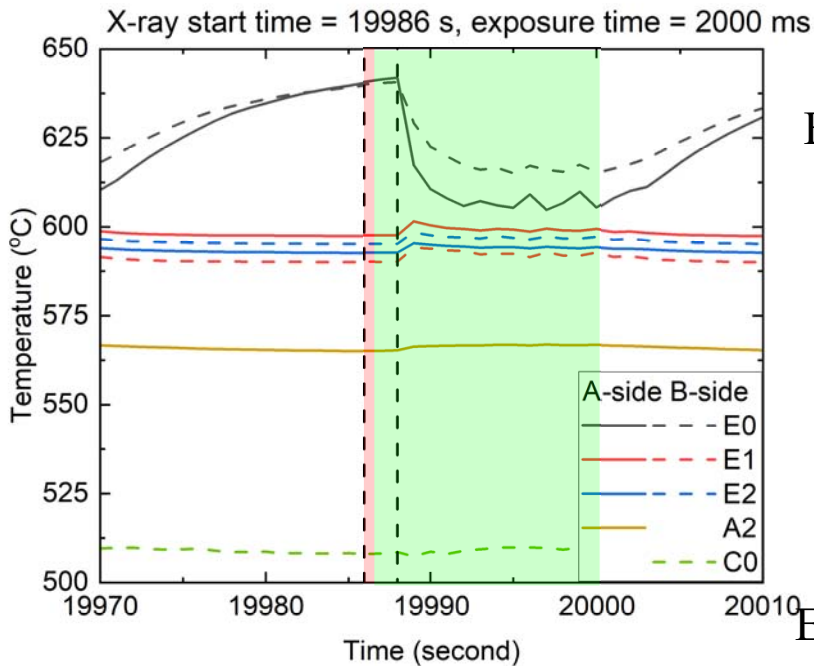
HP-b (102% FR):

- Inclined condition (45°)
- Primary channel: $m_a = 7$ g/s
- Auxiliary channel: $m_w = 10$ g/s

45 deg

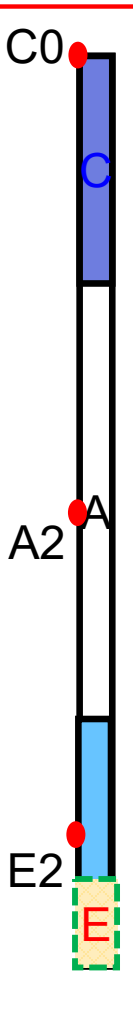
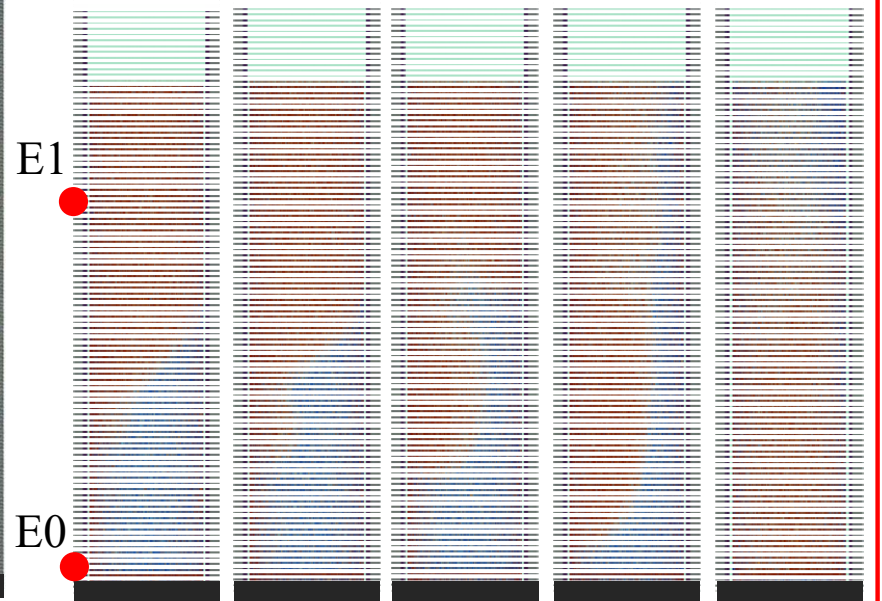
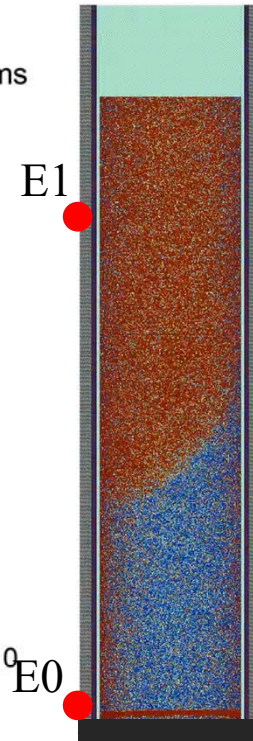


1/10 slow mo.



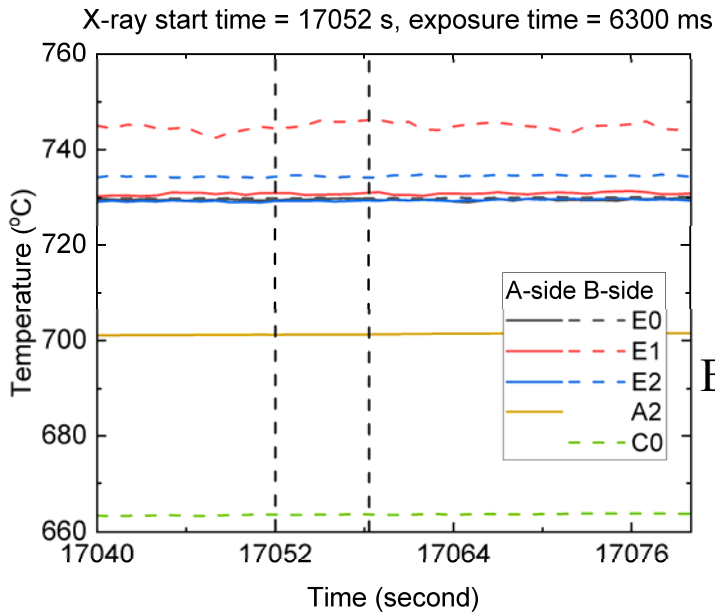
$T = 19886.11$ s

0 ms 10 ms 25 ms 70 ms 200 ms



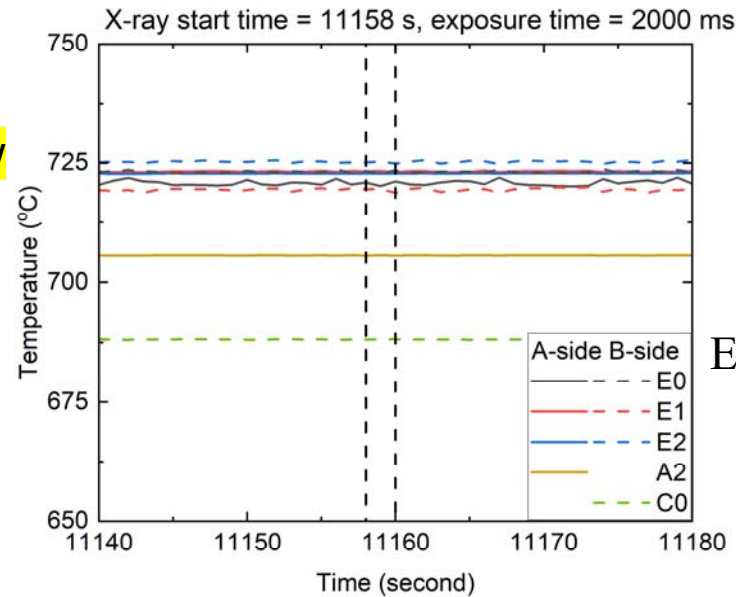
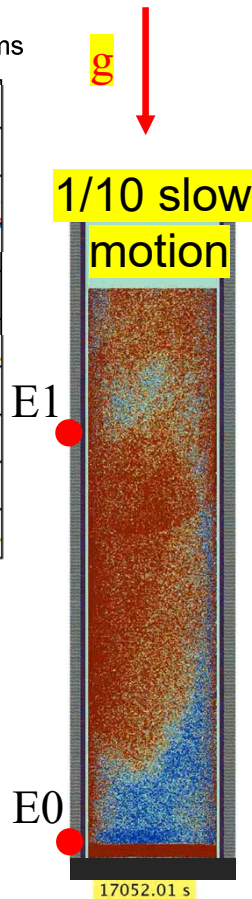
Result 2: Flow phenomenon

- Sequence of developed boiling phenomenon video synchronized with temperature data



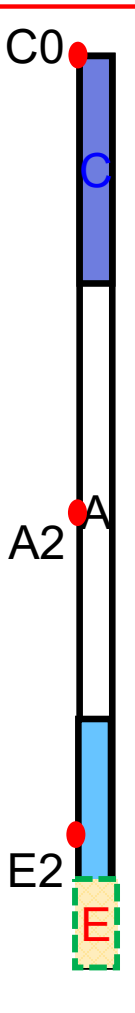
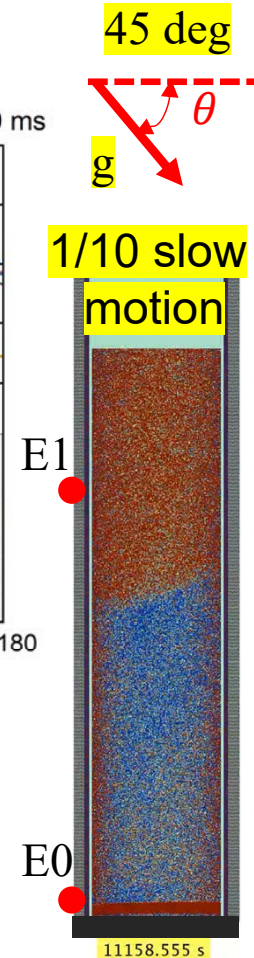
HP-b (102% FR):

- Vertical orientation (90°)
- Primary channel: $m_a = 7$ g/s
- Auxiliary channel: $m_w = 10$ g/s



HP-b (102% FR):

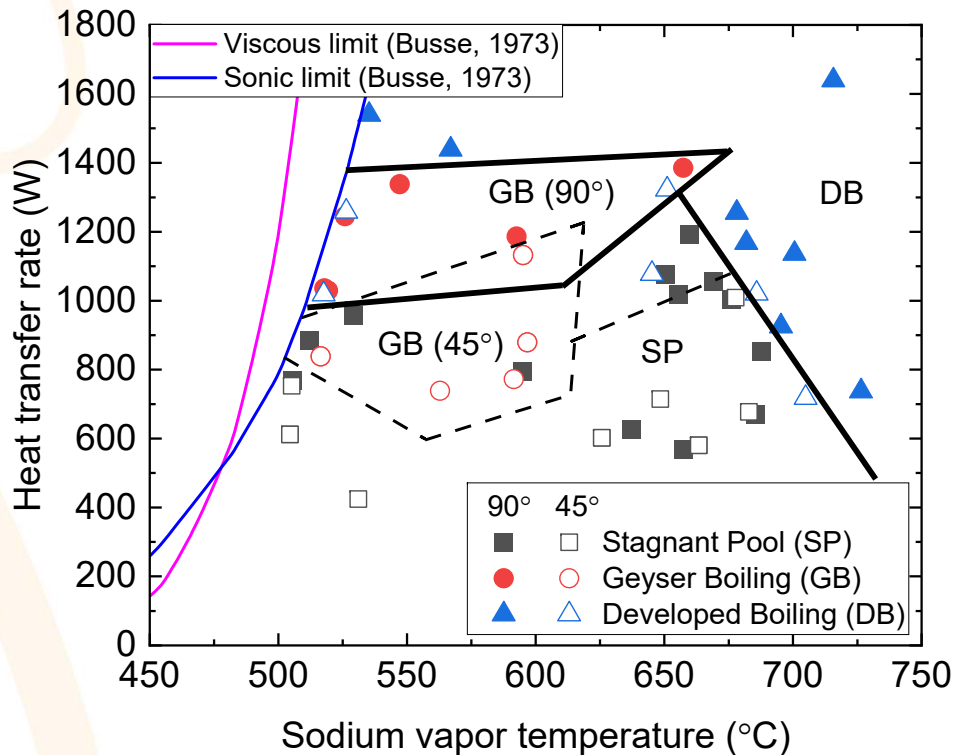
- Inclined condition (45°)
- Primary channel: $m_a = 1$ g/s
- Auxiliary channel: $m_w = 10$ g/s



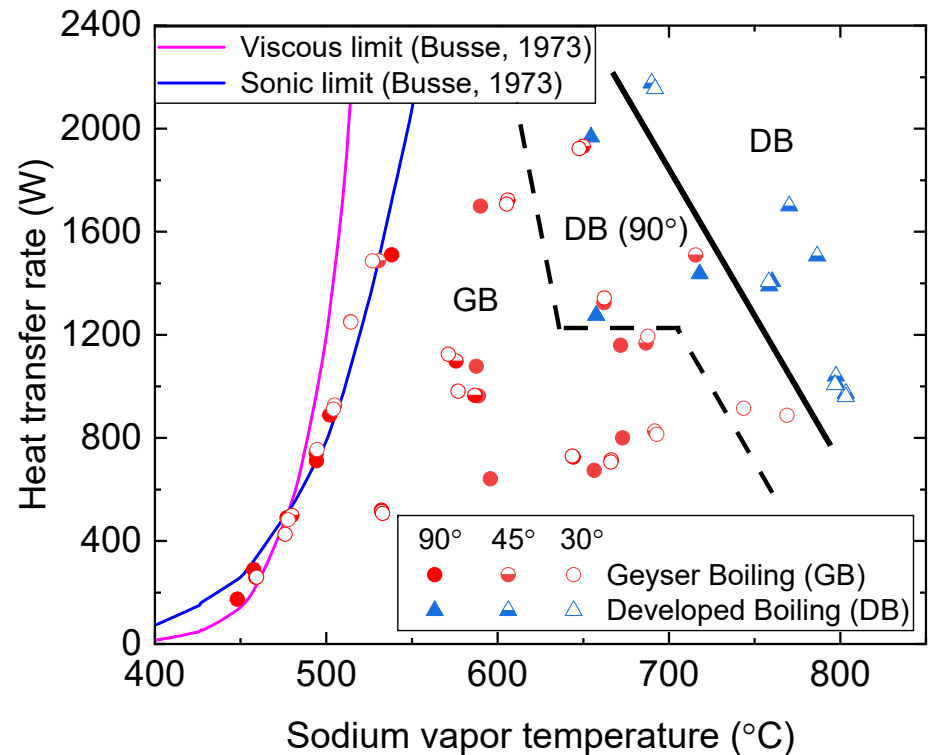
Result 2: Flow phenomenon

- Boiling flow regime map

- The initiation of boiling phenomena is closely related to the sodium filling ratio in the heat pipe and the orientation of heat pipe.



<HP-b w/ FR = 102%>



<HP-c w/ FR = 172%>

Result 3: Effect of parameters

- Effect of parameters

- Initial conditions

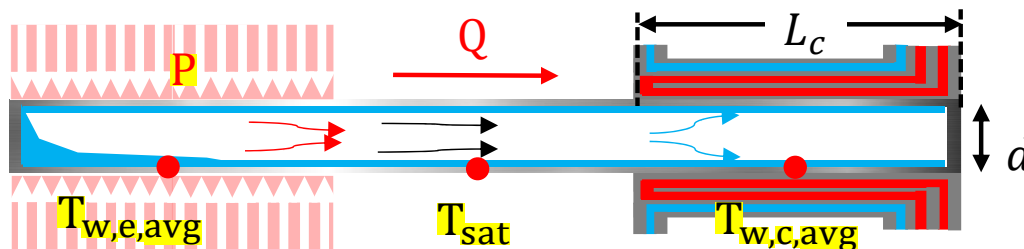
- Inclination angles: 0, 15, 30, 45, 60, 75, and 90°
 - Three heat pipes with sodium filling ratio of 67%, 102%, and 172%
 - Initial sodium location

- Boundary conditions

- Input power: 500 W, 800 W, 1100 W,...(Threshold $T_{w,max} \leq 900^{\circ}\text{C}$)
 - Five condenser conditions

- Data reduction

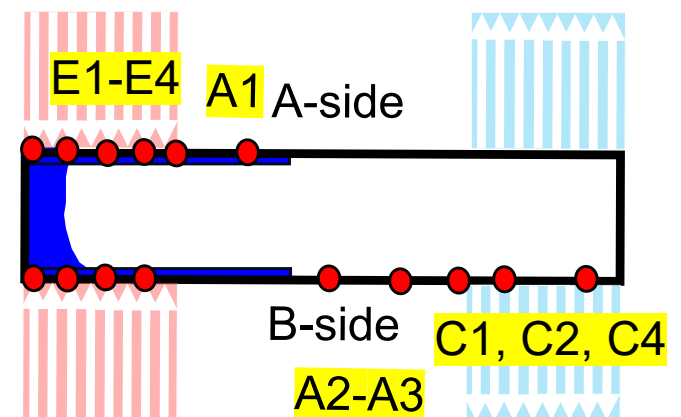
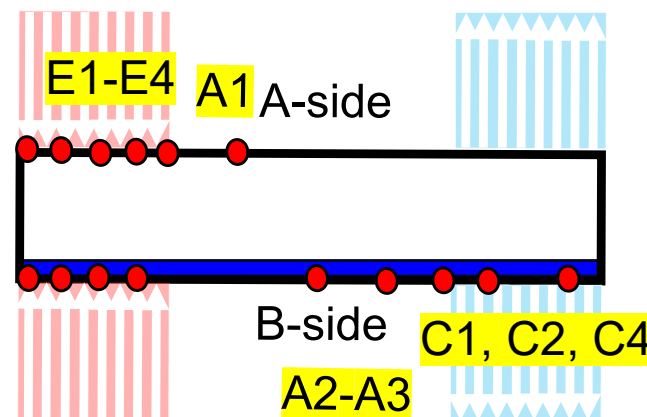
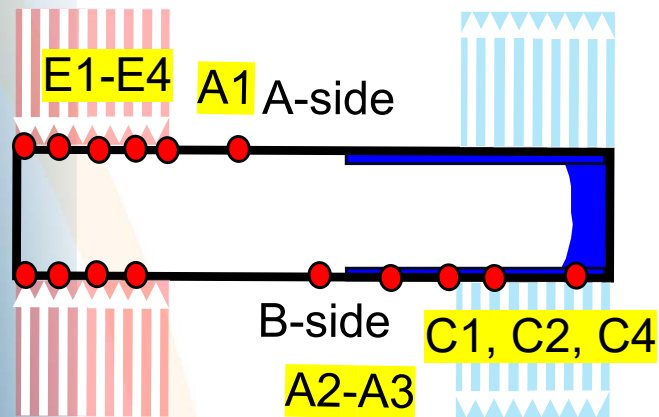
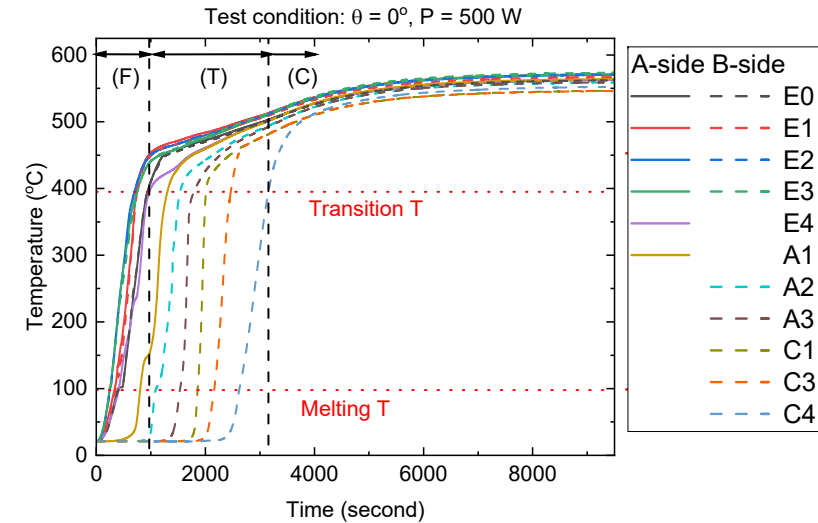
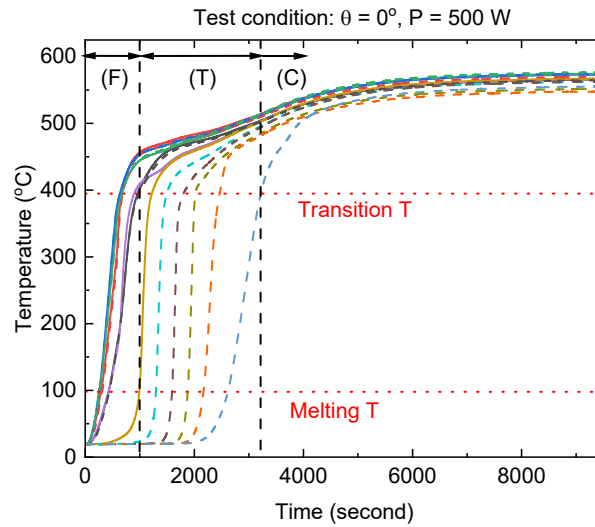
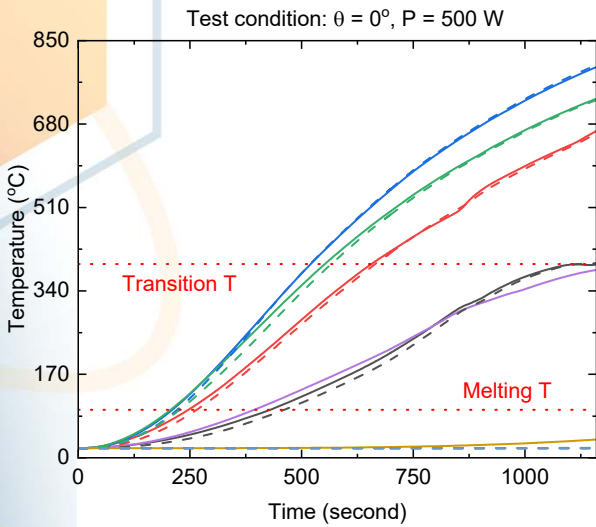
- Heat transfer rate of heat pipe: $Q = c_{p,P}\dot{m}_P(T_{fP2} - T_{fP1}) + c_{p,A}\dot{m}_A(T_{fA2} - T_{fA1})$
 - Heat transfer coefficient of heat exchanger: $h_{hx} = \frac{c_{p,P}\dot{m}_P(T_{fP2} - T_{fP1})}{[\pi d L_c (T_{w,c,avg} - T_{f,p,avg})]}$



Primary coolant: $T_{fP1,2}, c_{p,P}, \dot{m}_P$
 Auxiliary coolant: $T_{fA1,2}, c_{p,A}, \dot{m}_A$

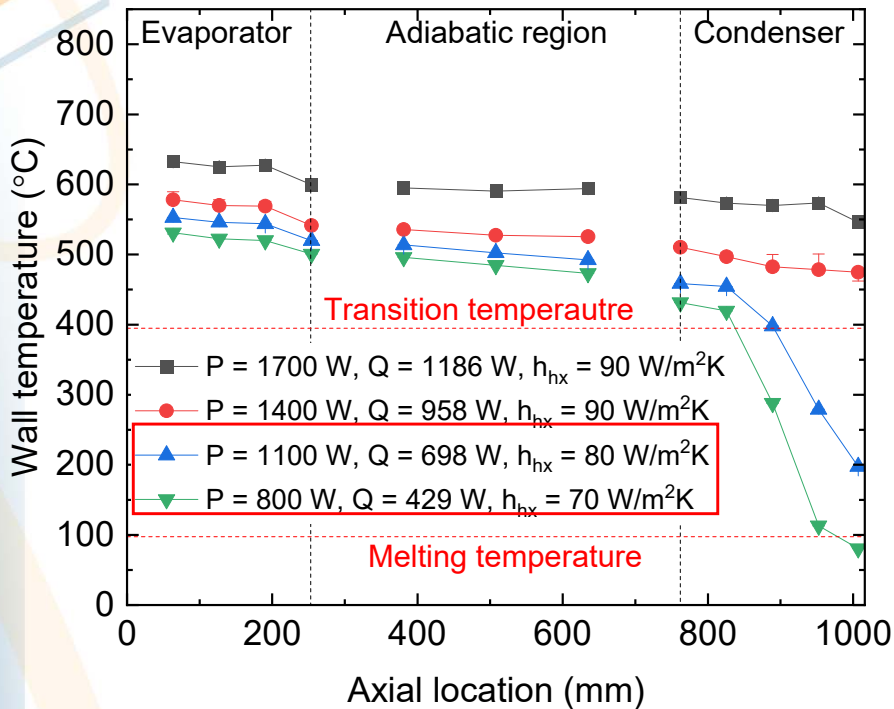
Result 3: Effect of parameters

- Effect of initial sodium location on startup characteristics (HP-b)

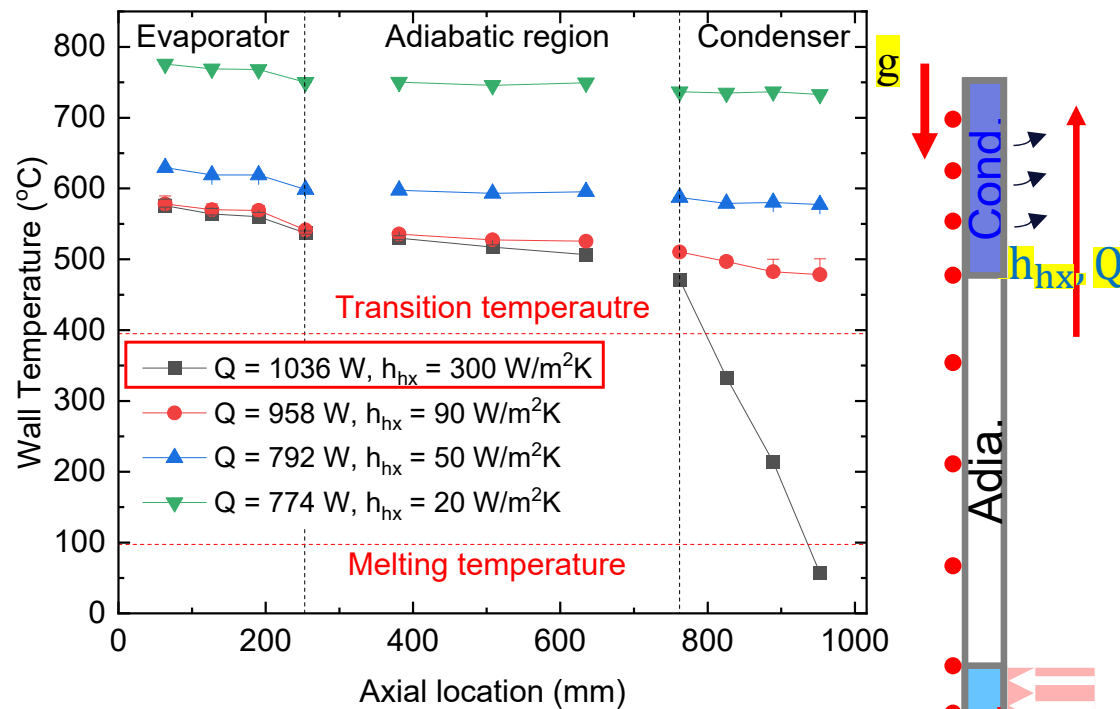


Result 3: Effect of parameters

- Effect of boundary conditions (input power and cooling intensity)
 - The balance between the input power and the cooling intensity is important



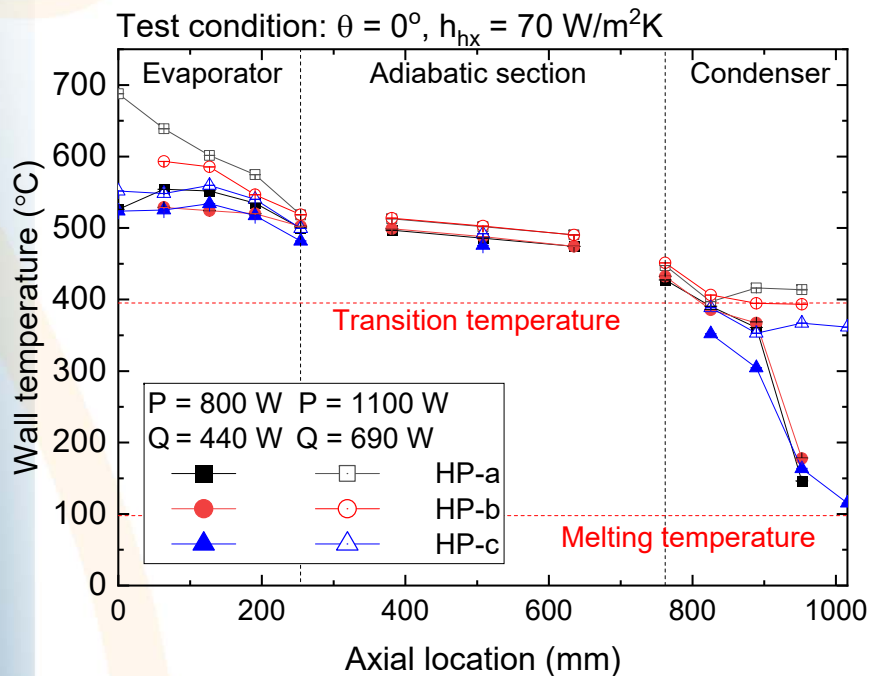
HP-b (102% FR), Vertical orientation ($\theta = 90^\circ$)
Cooling conditions: $m_a = 7$ g/s, $m_w = 10$ g/s



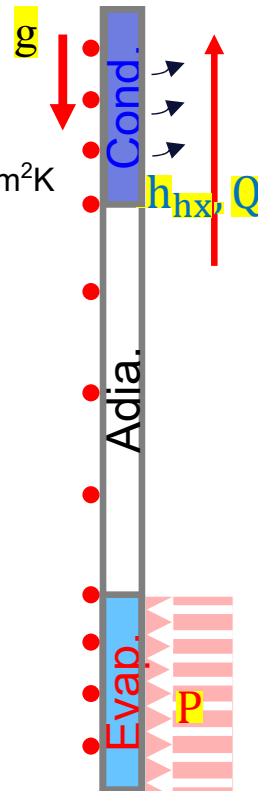
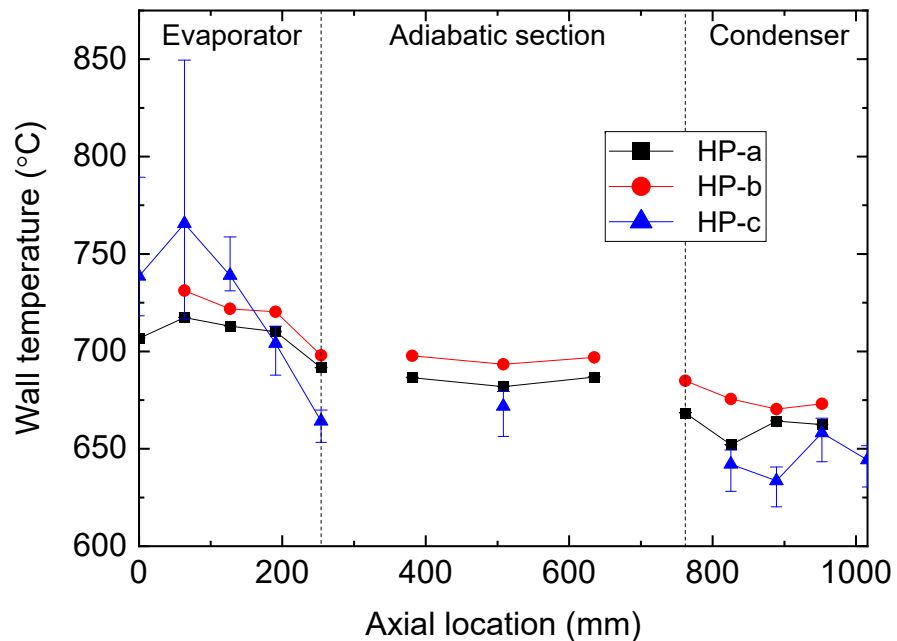
HP-b (102% FR), Vertical orientation ($\theta = 90^\circ$)
Input power: 1400 W, various cooling conditions

Result 3: Effect of parameters

- Effect of initial conditions (sodium filling ratio and inclination angle)
 - Overheat for heat pipe with small sodium filling ratio under horizontal orientation (dryout condition)
 - Severe temperature oscillation for heat pipe with large sodium filling ratio under vertical orientation (geyser boiling)



Test condition: $P = 1700 \text{ W}$, $Q = 1000 \text{ W}$, $\theta = 90^\circ$, $h_{hx} = 60 \text{ W/m}^2\text{K}$



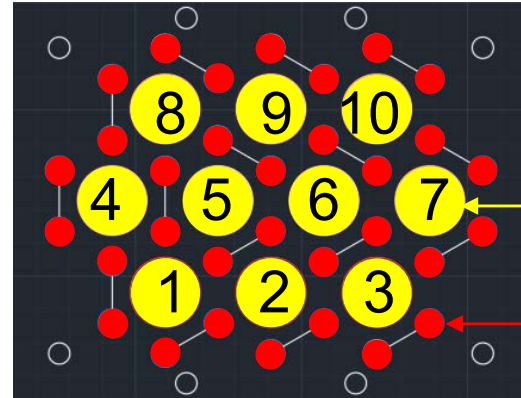
2. Sodium heat pipes bundle tests

Integral effect study with Michigan Sodium Heat pipe bundle test facility (MISOH2)

Experimental apparatus

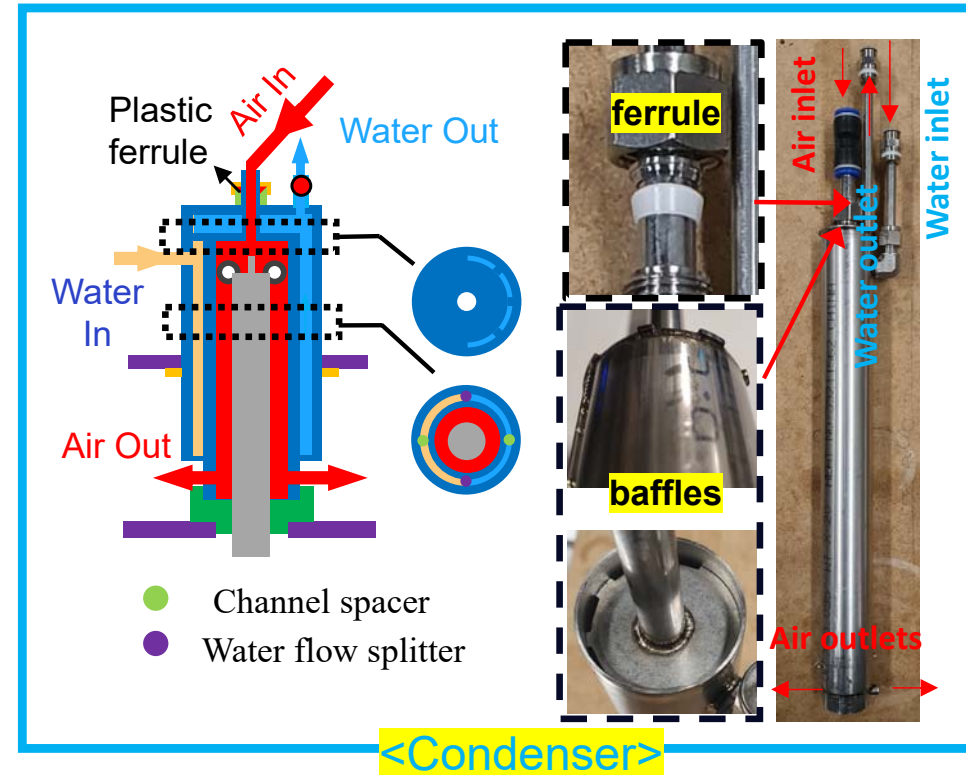
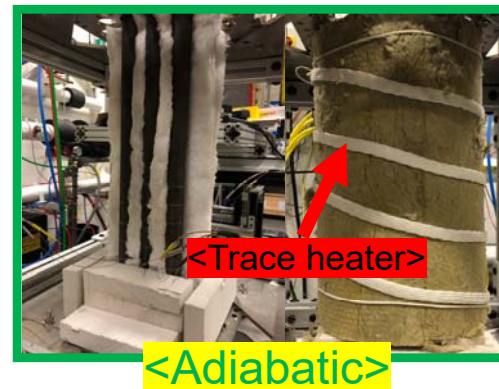
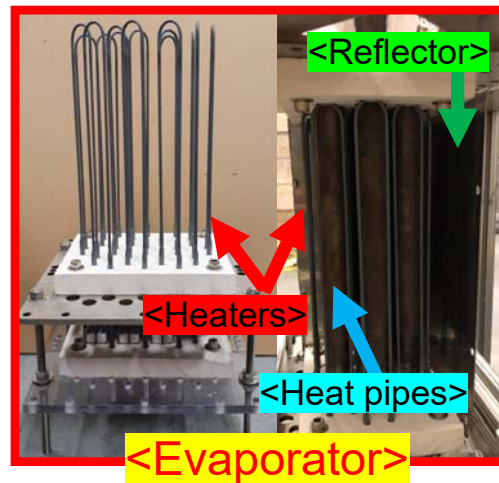
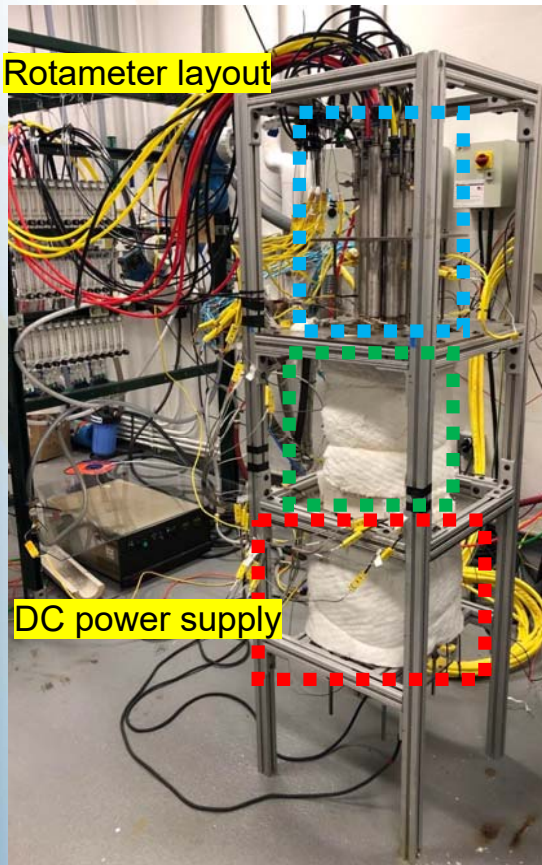
Spec. of the design

- Ten heat pipes array
- Triangular lattice
- P/D: 1.57
- 32 holes allocated for heating element(s)



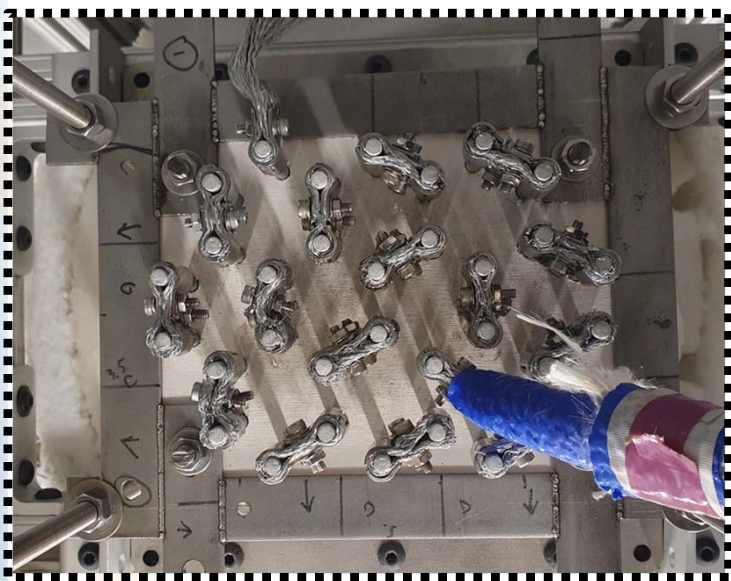
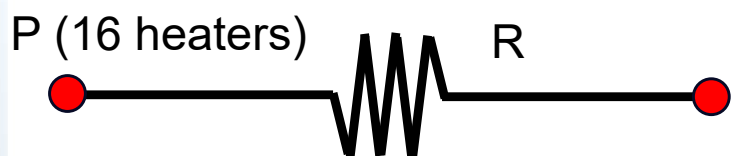
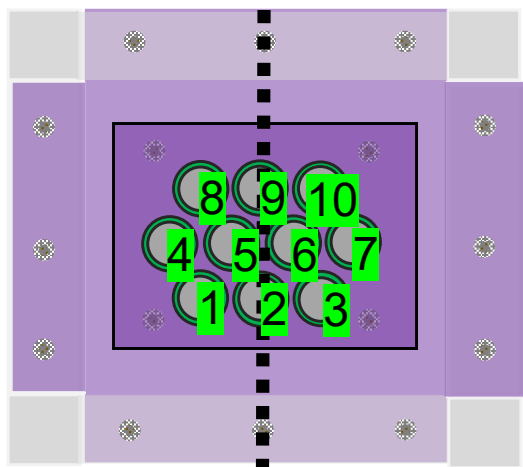
Heat pipe (22.1 mm ID)

Heating element



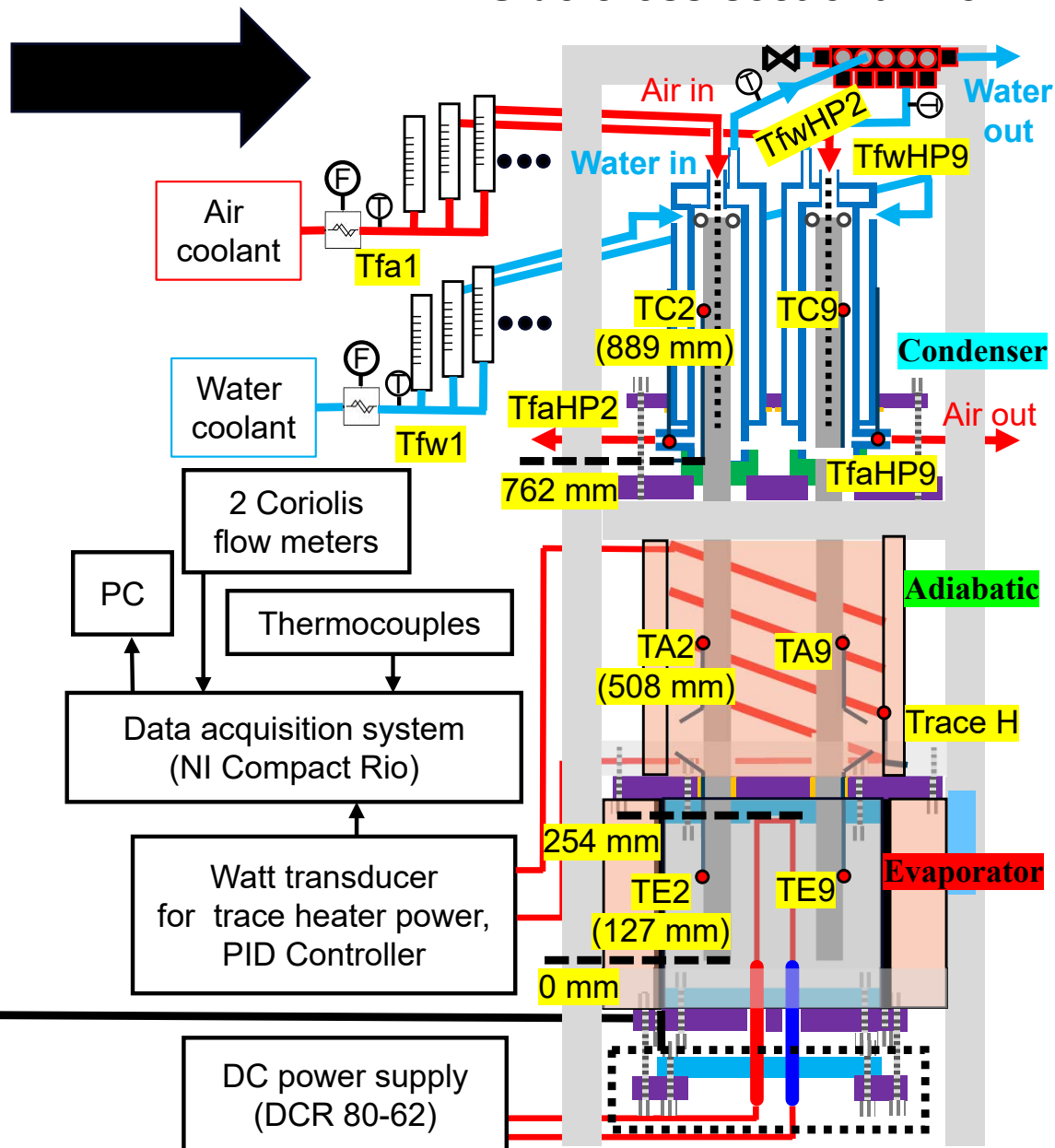
Experimental apparatus

<Top cross-sectional view>



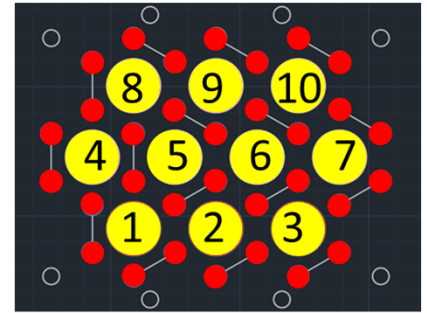
<Bridge connection for heaters>

<Side cross-sectional view>

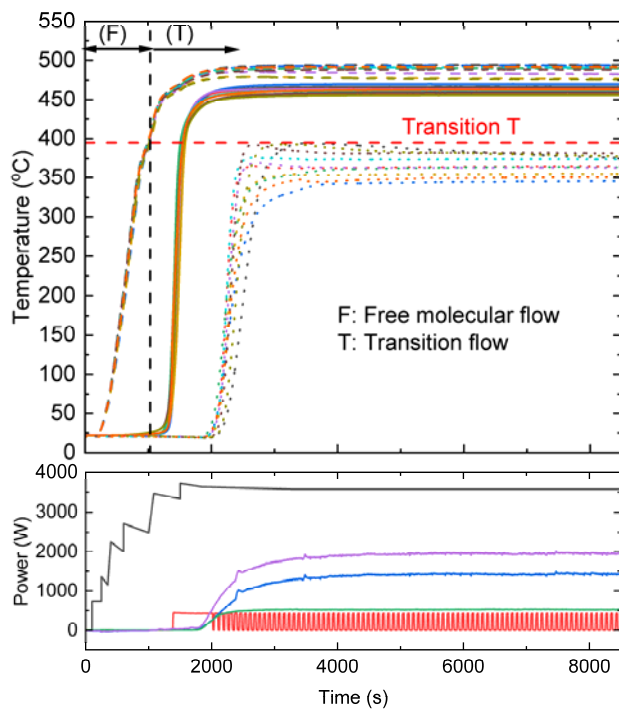


Preliminary Results

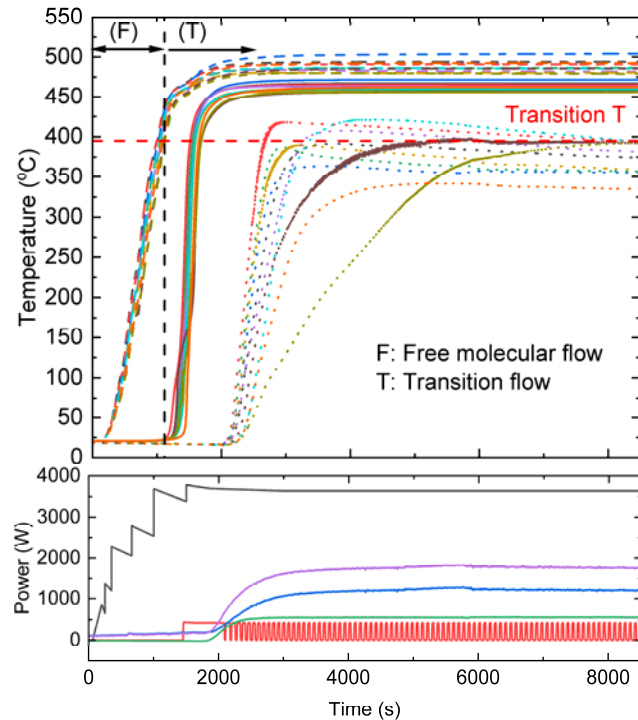
- Transient operation: startup process under uniform heating and cooling conditions



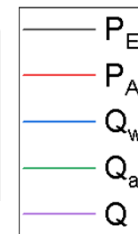
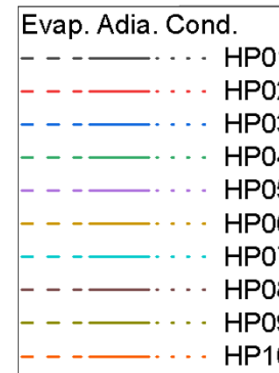
<HPs layout>



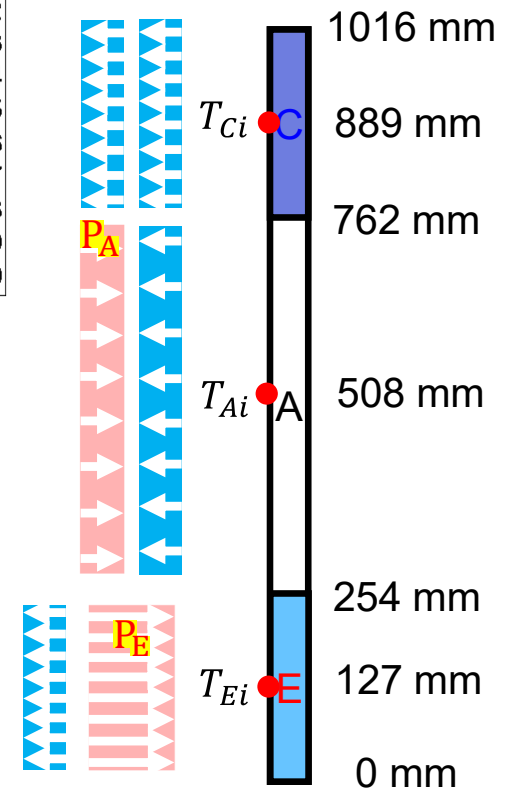
<Vertical orientation>



<Horizontal orientation>



$$Q_a + Q_w = Q$$



<HPi>

- HPs (FRs ranging from 90% - 110%)
- Condenser condition: $m_{a,i} = 0.5 \text{ g/s}$, $m_{w,i} = 5 \text{ g/s}$

Conclusion and Future Works

- Conclusion

- Separate effect of single sodium heat pipe

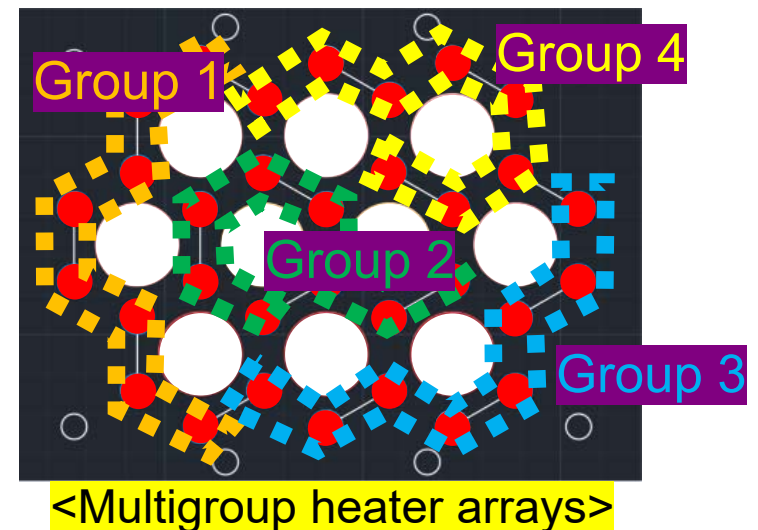
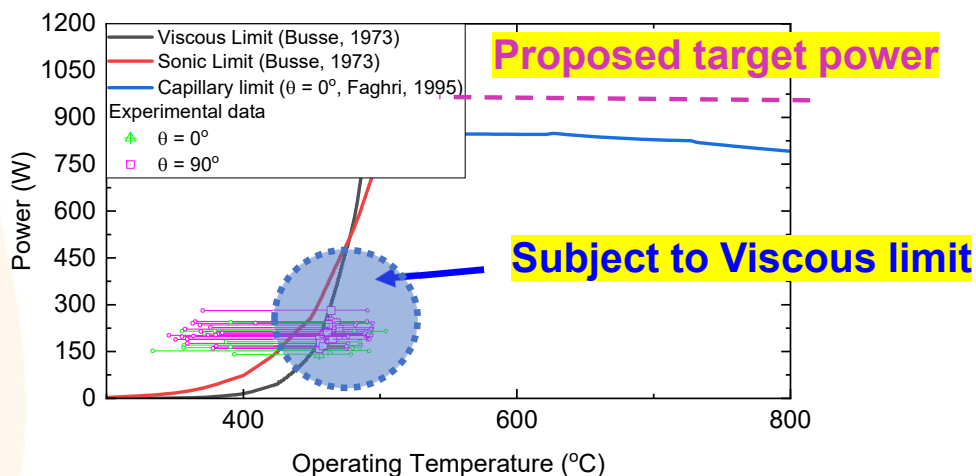
- Boiling region of heat pipes with different sodium filling ratio under different inclination angles was identified.
 - Heat pipe with high sodium filling ratio experienced significant temperature oscillation under large inclination angles.
 - Heat pipe with low sodium filling ratio experiences dryout conditions under horizontal orientation.

- Integral effect of heat pipes bundle

- Ten heat pipes hexagonal lattice array with individual variable heat exchangers were designed and constructed (MISOH2 test facility)

- Future works

- Power extension for MISOH2 test facility





Thank you For your attention Questions

Photo by Joseph Xu, College of Engineering